



IES/GATE

CIVIL ENGINEERING

VOLUME – III

O.C.F., Foundation, PERT - CPM



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O.C.F., Foundation, PERT - CPM

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OPEN CHANNEL FLOW

Chapter 1 - Introduction

| Pipe flow | open channel flow |
|--|---|
| ① Flow takes place due to difference in pressure | ① Flow takes place due to <u>gravity</u> |
| ② C/s is generally round | ② It can be triangular, trapezoid, rectangular etc. |
| ③ H.G.L. lies above the top surface of the water | ③ H.G.L. coincides with the top surface of the water. |
| ④ Reynold's Number is used | ④ Froude's number is used. |

Classification of channels :-

① natural and artificial channels

eg. River - natural

Canal - artificial

② Prismatic and non-prismatic channel

If the shape, C/s and bed slope is constant throughout

then channel is prismatic otherwise non-prismatic.

③ Rigid & mobile boundary channels

⇒ Rigid boundary channel

① Boundaries are not deformable.

No Sedimentation and erosion takes place.

1^o of freedom is there, i.e. depth of flow. (only depth can change)

MOBILE BOUNDARY CHANNELS

1) Boundaries are deformable

2) Sedimentation and erosion takes place

3) 4^o of freedom is there (all can change)

→ depth of flow ✓

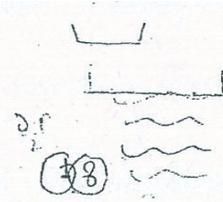
→ width of the channel ✓

→ Bed slope ✓

→ layout ✓

TYPES OF FLOW

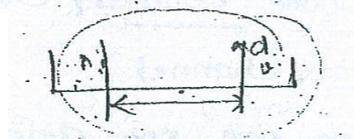
1) Steady and Unsteady flow :-



If all the flow parameters such as depth of flow, velocity, discharge etc. is constant w.r.t time then flow is steady otherwise unsteady.

2) uniform and non-uniform flow :-

If all the flow parameters w.r.t space is constant then flow is uniform otherwise non-uniform.



③ Laminar and turbulent flow :-

| Pipe flow | O.C.F |
|--------------------|------------------------------|
| $Re < 2000$ | $Re < 500$ Laminar |
| $2000 < Re < 4000$ | $500 < Re < 2000$ Transition |
| $Re > 4000$ | $Re > 2000$ Turbulent. |

④ Critical, Sub-Critical and Supercritical flow :-

$$Fr = \sqrt{\frac{F_i}{F_g}}$$

$$F_i = ma$$

$$F_g = mg$$

$$F_i = \rho Av^2$$

$$F_g = \rho ALg$$

$$Fr = \sqrt{\frac{\rho Av^2}{\rho ALg}}$$

$$Fr = \frac{V}{\sqrt{Lg}} \quad (\text{pipe flow})$$

$$Fr = \frac{V}{\sqrt{gD}} \quad (\text{O.C.F})$$

$$Fr = 1 \quad (\text{critical flow})$$

$$Fr > 1 \quad (\text{Supercritical, Rapid, shooting \& Torrential flow})$$

$$Fr < 1 \quad (\text{Subcritical, Tranquil \& streaming flow})$$

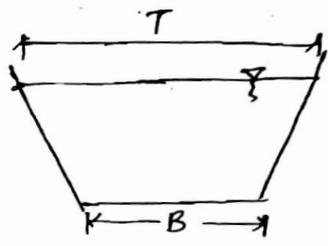
→ Critical, Subcritical, Supercritical flow :-

| | | |
|-----------------|--------------------|----------------------|
| <u>Critical</u> | <u>Subcritical</u> | <u>Supercritical</u> |
| → $F_r = 1$ | $F_r < 1$ | $F_r > 1$ |

Where, $F_r = \text{Froude's number}$

$$F_r = \frac{V}{\sqrt{\frac{gA}{T}}}$$

$V \rightarrow$ Avg. velocity of flow
 $A \rightarrow$ Area of CS of flow
 $T \rightarrow$ Top width & $g = \text{acc. due to gravity}$



| | | |
|-----------------------------------|-----------|-----------|
| → $V = V_c = \sqrt{\frac{gA}{T}}$ | $V < V_c$ | $V > V_c$ |
| → $y = y_c$ | $y > y_c$ | $y < y_c$ |

→ Denominator of Froude number represents a speed with which the disturbance created to flow travels in still water. i.e, "Celerity (C)".

$$\therefore \text{Celerity } (C) = \sqrt{\frac{gA}{T}}$$



$$V_{\text{wave/ground (US)}} = V_{\text{wave/water (US)}} + V_{\text{water/ground (US)}}$$

$$\therefore V_{\text{wave/ground (US)}} = C - V$$

[∵ US → upstream]

At low flow velocity
 $F_r < 1$
 $C > V$
 $C - V > 0$

→ At low flow velocity, $F_r < 1$ and a small disturbance to the flow will cause disturbance wave which travels upstream with a velocity of $(C - V)$ w.r.t a stationary observer.

→ Due to the upstream movement of waves upstream conditions gets affected, In case of Subcritical flow, condition upstream is affected by condition at downstream. Hence downstream section is taken as "Control section".

At high flow velocity $F_r > 1$
 $V > C$
 $C - V < 0$

Thus upstream flow velocity of waves ($C - V$) will become negative. i.e. the disturbance waves will not ^{travel} upstream, It will travel downstream with a velocity of $V - C$. Hence flow condition Down stream would be affected. Thus

Supercritical flow has upstream control.

ste:-

Subcritical flow has downstream control while supercritical flow has upstream control.

* Subcritical flow → Down stream Control

* Supercritical flow → Up stream control



→ When $F_r = 1$, i.e. flow is critical, the disturbance velocity $C - V = 0$
 i.e. the disturbance wave will not travel at all.

IE:-

A wide channel is 1m deep and has a velocity of flow $V = 2.13$ m/sec, if a disturbance is caused and elementary wave can travel upstream with a velocity of —?

- a) 1 m/sec b) 2.13 m/sec c) 3.13 m/sec d) 5.26 m/sec

Sol:- $V = 2.13$ m/sec, 1 wide channel (1m) → Rectangular channel

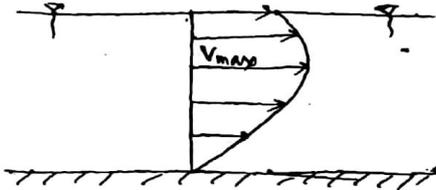
$y = 1$ m

W.K.T, $C = \sqrt{\frac{gA}{T}}$

∴ celerity, $C = \sqrt{\frac{g \times y^3}{y}} = \sqrt{gy} = \sqrt{9.81 \times 1} = 3.13$ m/sec

$$\begin{aligned}
 \therefore V_{\text{wave/ground}} &= C - V \\
 &= 3 - 13 - 2 = 13 \\
 &= 1 \text{ m/sec}
 \end{aligned}$$

→ Velocity Distribution :

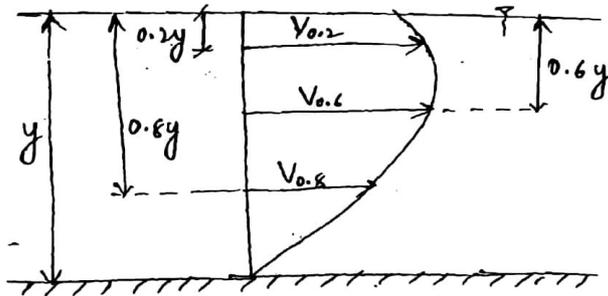


→ Reduction (or) Dip in the velocity is because of secondary currents, which is a function of aspect ratio.

$$\therefore \text{Aspect ratio} = \frac{\text{Depth}}{\text{width}}$$

Note :

→ if Aspect ratio is large depth at which max. velocity will occur is deeper.

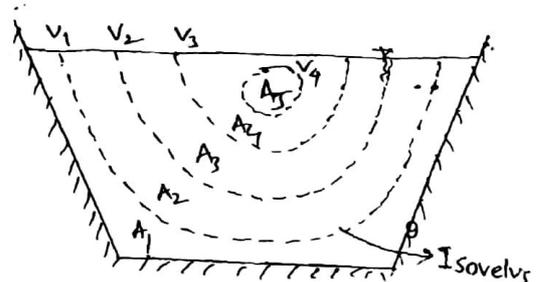


$$\begin{aligned}
 V_{\text{avg}} &= \frac{V_{0.2} + V_{0.8}}{2} \\
 \text{(or)} \\
 V_{\text{avg}} &= V_{0.6} \text{ (less reliable)}
 \end{aligned}$$

V_{avg} → Average Velocities.

→ Isovels :

→ Contour of equal velocity is called "Isovels".



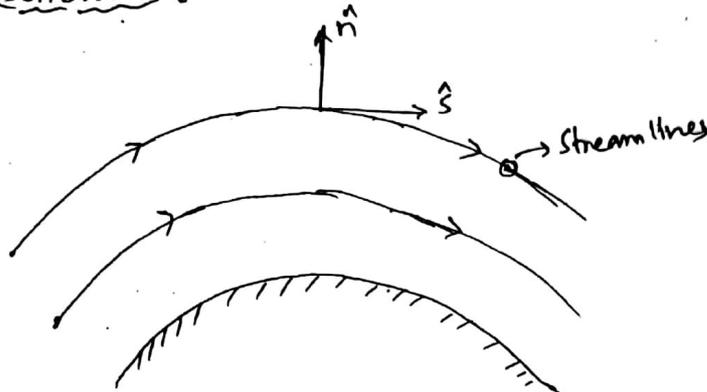
→ The avg. velocity V_s ^{also} sometimes calculated as,

$$\boxed{V_{avg} = K V_{surface}}, [0.8 < K < 0.95]$$

Note:-

→ Solve for the better approach

→ Pressure Distribution :-



By Euler's formula ;

$$-\frac{\partial}{\partial n} (P + \gamma z) = \rho a_n$$

a_n → Normal acceleration

$$a_n = \frac{V^2}{r}, \quad \gamma = \rho g$$

P → Gauge pressure, z = Datum (elevation)

→ if stream lines are straight lines then the normal acceleration (a_n) would be

zero, because, $r \rightarrow \infty \Rightarrow a_n \rightarrow 0$:

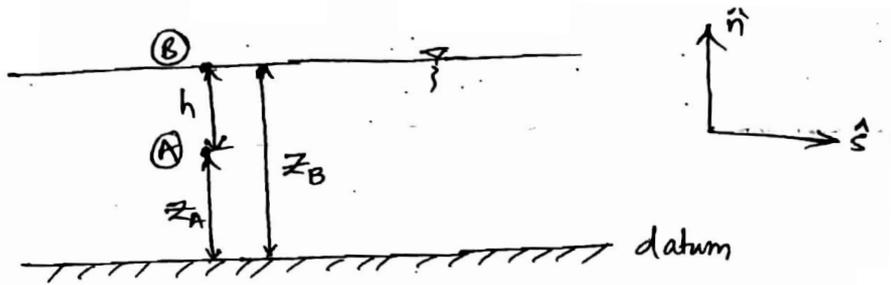
$$\rightarrow \frac{-\partial (P + \gamma z)}{\partial n} = 0$$

$$P + \gamma z = \text{constant}$$

$$\Rightarrow \text{Piezometric head} = \left(\frac{P}{\gamma} + z \right) = \text{constant}$$

→ Hence the Piezometric head will become constant i.e. Pressure distribution is "Hydro static".

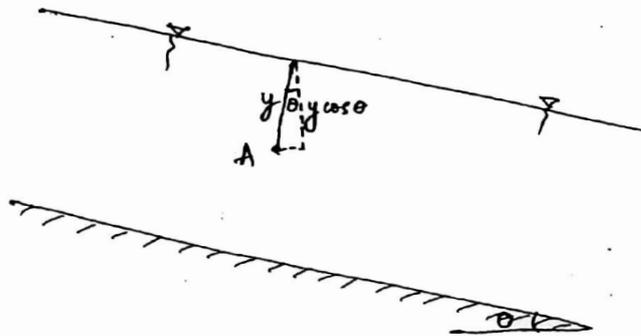
Proof:-



$$\Rightarrow P_A + \gamma_w z_A = P_B + \gamma_w z_B \quad [\text{Here, } P_B = 0 \text{ (atmospheric pressure)}]$$

$$\Rightarrow P_A = \gamma_w (z_B - z_A) = \gamma_w h.$$

Non-Curvilinear flow \rightarrow



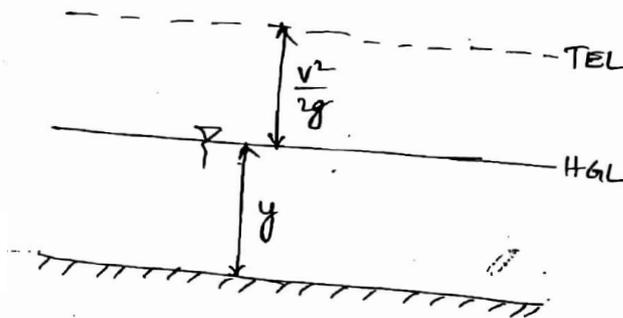
$$P_A = \gamma_w (y \cos \theta)$$

when bed slopes are small

$$\Rightarrow \theta \approx 0 \Rightarrow \cos \theta \approx 1$$

$$\therefore \boxed{P_A = \gamma_w y}$$

\rightarrow



HGL \rightarrow Hydraulic gradient line
 TEL \rightarrow Total Energy line

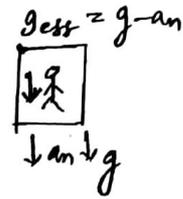
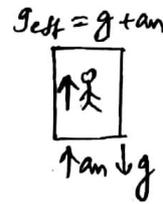
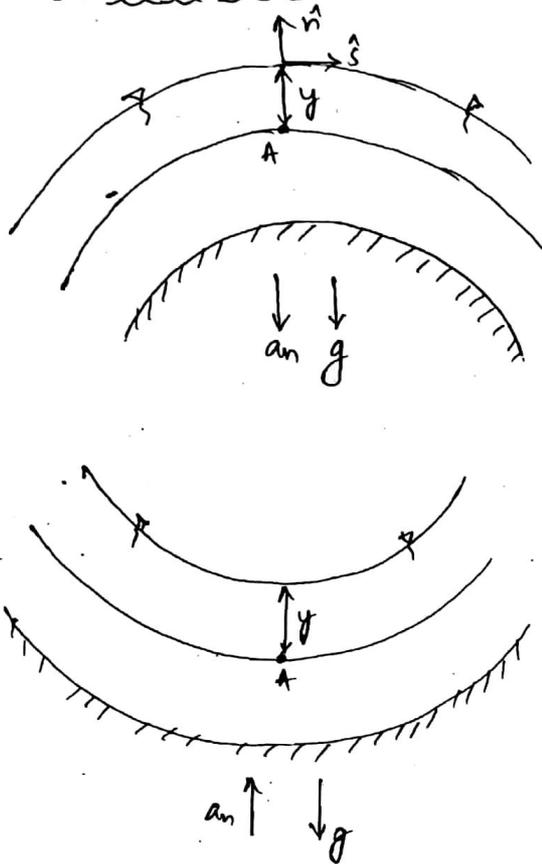
\rightarrow When $(\frac{P}{\gamma_w} + z)$ is plotted along the length we get HGL

\rightarrow When $(\frac{P}{\gamma_w} + z + \frac{v^2}{2g})$ is plotted along the length we get TEL

\rightarrow For smaller slopes HGL will coincide with free surface and for large slopes HGL will be below the free surface. ($y \cos \theta < y$).

For smaller slopes HGL will coincide with the free surface and for the large slope HGL will be below the free surface ($\because y \cos \theta < y$) [Repeated].

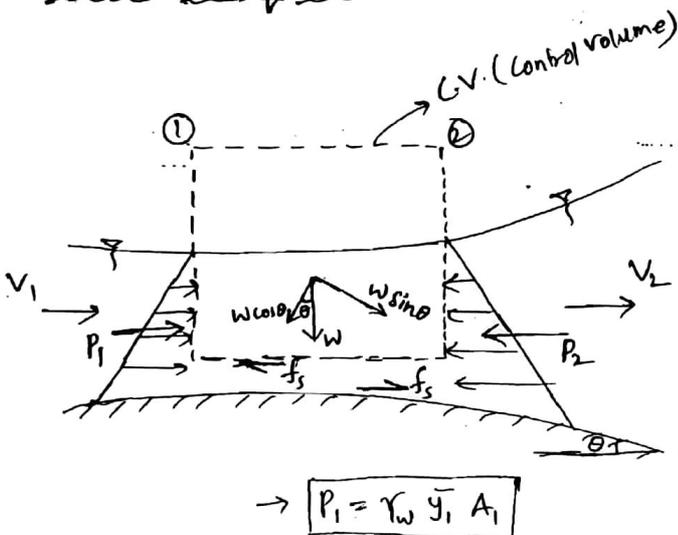
Curvilinear flow :-



$g_{eff} = g - an$
 $P_A = \rho g_{eff} y$

$g_{eff} = g + an$
 $P_A = \rho g_{eff} y$

Momentum equation :-



$\rightarrow P_1 = \gamma_w \bar{y}_1 A_1$

Where, \bar{y}_1 = depth of C.G. of area from free surface.

Net force on C.V. = $P_1 - P_2 + W \sin \theta - f_s = M_2 - M_1$

$P_1, P_2 \rightarrow$ Pressure force

$M \rightarrow$ Momentum flux ($\text{kg} \cdot \text{m/s}^2$)

$M_1 = \rho Q V_1$ & $M_2 = \rho Q V_2$

Rate of change of momentum = $M_2 - M_1$

$P_1 = \frac{1}{2} (\gamma_w y) \times y \times B$

$P_2 = \gamma_w \times \frac{1}{2} \times y \times B$

→ Thus pressure force on an area is found out by multiplying that pressure at the C.G. of the area with the complete area.

Note:-

→ In actual form for steady flow the momentum eqⁿ is written as,

"Net external force acting in x-direction on the Control volume = Momentum flux going out of control volume in x-direction - Momentum flux coming inside the control volume in x-direction".

→ Specific force :-

$$\boxed{\text{Specific force} = \frac{P+M}{\gamma_w}}$$

where, P = pressure force

M = momentum flux ($\rho Q V_1$)

→ Specific force is constant for a horizontal frictionless channel

→ From momentum eqⁿ,

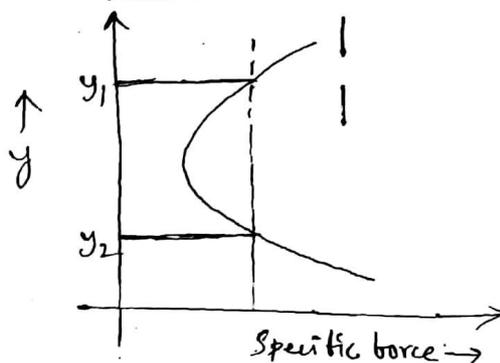
$$P_1 - P_2 = M_2 - M_1$$

$$\frac{P_1 + M_1}{\gamma_w} = \frac{P_2 + M_2}{\gamma_w} = \text{Specific force}$$

∴ Specific force = constant.

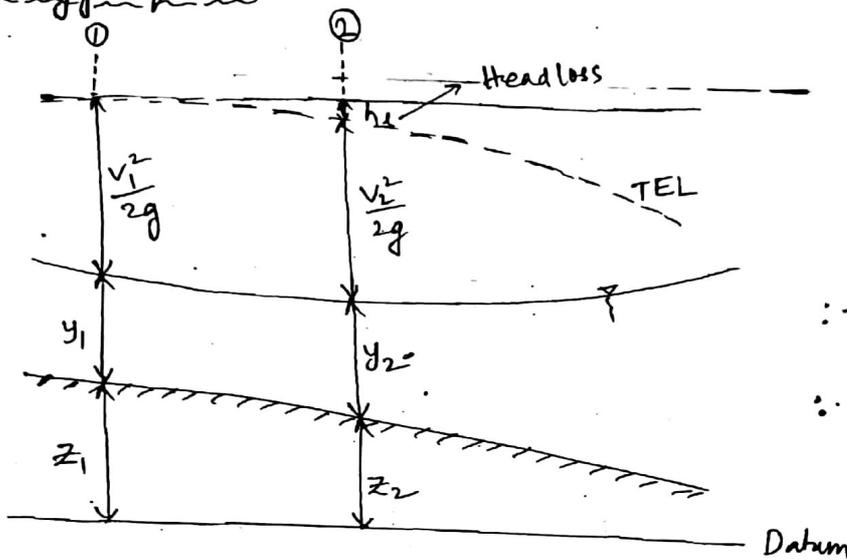
→ In the Analysis of RVF, friction doesnot play an important role, Hence for smaller slopes of bed RVF Analysis is done by making the specific force to be constant.

→ At a constant specific force, we have two possible depth of flow and they are called as "Conjugate depth or Sequent depth".



y_1 & y_2 → Conjugate depths or Sequent depth

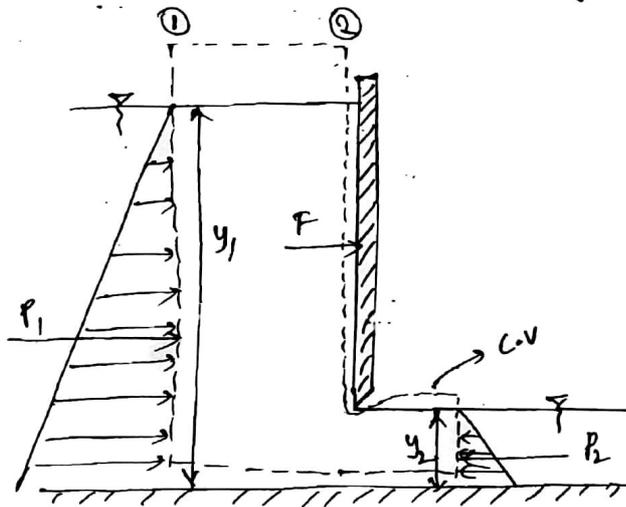
→ Energy Equation :-



$$\rightarrow y_1 + z_1 + \frac{v_1^2}{2g} = y_2 + z_2 + \frac{v_2^2}{2g} + h_L$$

→ Every term of the eqⁿ is representing "Energy Head", i.e., "Energy per unit wt"

∴ Find the force per unit width on the sluice gate due to water forces. neglect energy loss & consider channel to be rectangular.



Sol: By applying momentum eqⁿ on C.V,

$$P_1 - P_2 - F = M_2 - M_1$$

P → Pressure force, M → Momentum

$$P_1 = \gamma_w \bar{y}_1 A_1$$

$P_1 = \gamma_w \cdot \frac{y_1}{2} \times (y_1 \times 1)$ ----- Considering unit width inside

$$P_1 = \frac{\gamma_w y_1^2}{2}, \quad P_2 = \frac{\gamma_w y_2^2}{2}$$

$$\rightarrow M_1 = P_1 \cdot y_1 = P_1 \times \frac{2}{y_1} = \frac{P_1^2}{y_1}$$

$$\rightarrow M_2 = \frac{P_2^2}{y_2}$$

$$\Rightarrow \frac{\gamma_w}{2} (y_1^2 - y_2^2) - F = P_1^2 \left(\frac{1}{y_2} - \frac{1}{y_1} \right)$$

$$\therefore F = \frac{\gamma_w}{2} (y_1^2 - y_2^2) - P_1^2 \left(\frac{1}{y_2} - \frac{1}{y_1} \right) \rightarrow \textcircled{1}$$

By applying energy equation,

$$y_1 + z_1 + \frac{v_1^2}{2g} = y_2 + z_2 + \frac{v_2^2}{2g} + h_L$$

$$y_1 + \frac{v_1^2}{2g} = y_2 + \frac{v_2^2}{2g}$$

(By continuity equation, $v_1 (y_1 \times 1) = v_2 y_2 = Q$)

$$\Rightarrow y_1 + \frac{Q^2}{2g y_1^2} = y_2 + \frac{Q^2}{2g y_2^2}$$

$$\frac{Q^2}{2g} \left(\frac{1}{y_2^2} - \frac{1}{y_1^2} \right) = y_1 - y_2$$

$$Q^2 = \frac{2g y_1^2 y_2^2}{y_1 + y_2}$$

By substituting in eqⁿ ①,

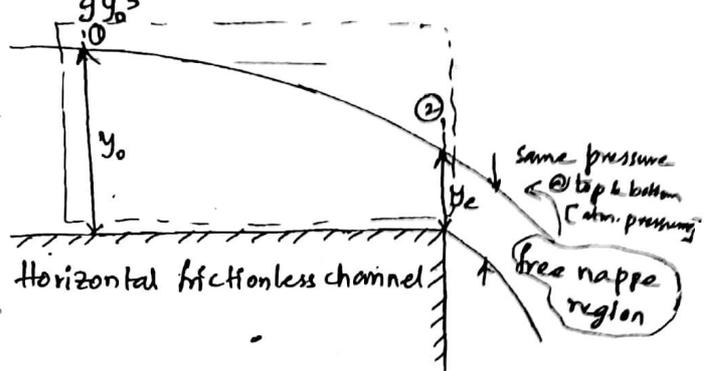
$$F = \frac{\gamma_w}{2} (y_1^2 - y_2^2) - 2 \rho g \frac{y_1^2 y_2^2}{y_1 + y_2} \left(\frac{1}{y_2} - \frac{1}{y_1} \right)$$

$$F = \frac{\gamma_w}{2} (y_1 - y_2) \left(y_1 + y_2 - \frac{4 y_1 y_2}{y_1 + y_2} \right)$$

$$\therefore \boxed{F = \frac{\gamma_w (y_1 - y_2)^3}{2(y_1 + y_2)}} \quad (\text{remember})$$

→ The above eqⁿ is valid for a Rectangular channel when energy loss & friction is neglected.

∴ Prove that, $\frac{y_e}{y_0} = \frac{2F_0^2}{2F_0^2+1}$ & $F_0^2 = \frac{q^2}{gy_0^3}$



Sol: $F_r = \frac{V}{\sqrt{\frac{gA}{T}}} = \frac{Q/A}{\sqrt{\frac{gA}{T}}}$
 $\rightarrow F_r^2 = \frac{Q^2 T}{gA^3}$

for rectangular channel, $\Rightarrow F_r^2 = \frac{Q^2 B}{g(B^3)} = \frac{(Q/B)^2}{gy^3} = \frac{q^2}{gy^3}$

[q → discharge per unit width]

→ In the free nappe, Pressure through out the depth is Atmospheric.

→ Applying Momentum eqⁿ between ① & ②:

$$P_1 - P_2 = M_2 - M_1$$

$$\gamma_w \frac{y_0}{2} \times (y_0 \times B) = \rho Q v_e - \rho Q v_0$$

(By continuity eqⁿ, $B y_0 v_0 = B y_e v_e = Q$)

$$\frac{\gamma_w y_0^2 B}{2} = \frac{\rho Q^2}{B} \left(\frac{1}{y_e} - \frac{1}{y_0} \right)$$

$$\frac{gy_0^2}{2} = \frac{(Q/B)^2}{y_0} \left(\frac{y_0}{y_e} - 1 \right) \quad \dots \dots \dots (q = Q/B) \text{ \& } (\gamma_w = \rho g)$$

$$\frac{1}{\frac{2q^2}{gy_0^3}} = \left(\frac{y_0}{y_e} - 1 \right) \quad \dots \dots \dots (F_0^2 = \frac{q^2}{gy_0^3})$$

$$\Rightarrow \frac{y_0}{y_e} - 1 = \frac{1}{2F_0^2}$$

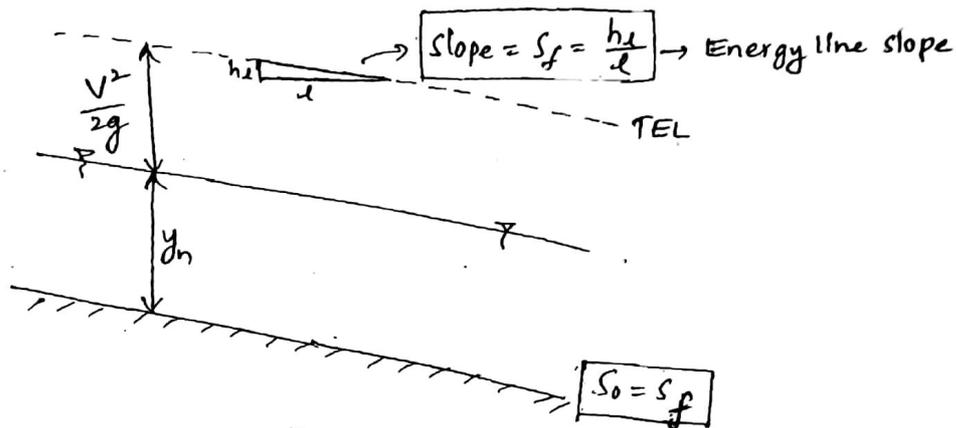
$$\Rightarrow \boxed{\frac{y_e}{y_0} = \frac{2F_0^2}{1+2F_0^2}}$$

(Proved)

Note: Generally eqⁿ used in the analysis of open channel flow are,
 i) Energy equation
 ii) Momentum equation
 iii) Continuity equation.

2. Uniform flow

- Uniform flow;
- Flow in an open channel is said to be uniform, if the depth of flow is same at every section of the channel, and this constant depth is called "normal depth of flow (y_n)".
- for uniform flow, bed slope, cross-section & surface roughness must remain constant thus the depth, velocity & discharge remains constant.
- In the analysis of uniform flow, flow is assumed to be steady.
- As the depth & velocity is same at every cross-section, the bed slope, water surface slope, Energy Line slope will all be same.



→ Formulae for determining velocity and discharge are;

1. Chezy's equation;

$$V = C \sqrt{RS}$$

V → Avg. velocity of flow

C → Chezy's constant

$$C = \sqrt{\frac{8g}{f}} \quad (f \rightarrow \text{friction factor})$$

$$R = \frac{A}{P} \quad (\text{hydraulic radius})$$

S → Energy line slope

Manning's Equation)

$$V = \frac{1}{n} R^{2/3} S^{1/2}$$

$$h_e = \frac{\rho L V^2}{2gD}$$

V = Avg. Velocity

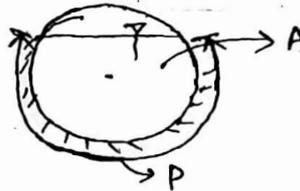
n = Manning's constant (or) roughness coefficient

$$R = \frac{A}{P}$$

A → Area of cs

P → Wetted Perimeter

S → Energy line slope.

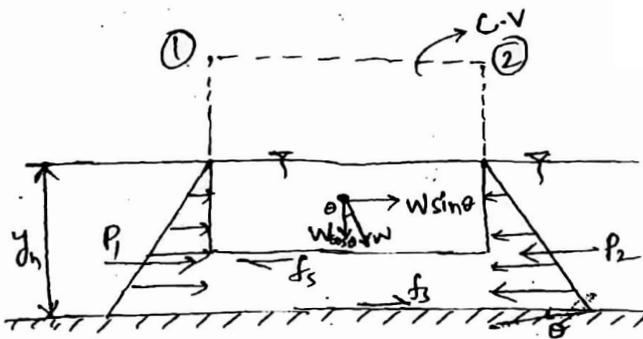


→ Energy line slope, However since in uniform ^{flow}, Bed slope, water surface slope, and energy line slope are equal so, $S = S_0$

note:

→ If "S" is used as " S_f " → we get actual depth of flow, Hence in varied flow actual depth of flow may not be equal to normal depth of flow.

Momentum equation of uniform flow ;



→ By momentum eqⁿ on C.V. (control volume)

$$P_1 - P_2 + W \sin \theta - f_s = M_2 - M_1$$

for uniform flow, $P_1 = P_2 = \gamma_w \cdot \bar{Y}_n \cdot A$

$$M_1 = M_2 = \rho Q V$$

$$\Rightarrow \boxed{f_s = W \sin \theta}$$

Energy derived from gravity = Energy loss due to friction.